#### **Inclined Road Forces**

Gradient Resistance (GR)

The gradient resistance (climbing resistance, inclined road force) depends on the angle of the road inclination and the weight of the car.

$$GR = W \sin\theta = mg \sin\theta$$

where:

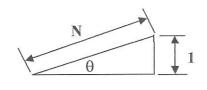
W= the car weight (N) = mg

 $\theta$  = the angle of road inclination

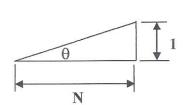
Road inclination (gradient):

The road gradient can be described as 1 in N, this description can be either:

a-



b-



The description (b) is not suitable incase of level road the gradient will be 1 in  $\infty$ , especially when using the computer. So, the description of the gradient will be as in (a), and the gradient will be  $(G = \sin\theta)$ . The gradient can be written in as a percentage  $(G = \sin\theta) = S\%$ .

\* For small angle ( $\theta$  (radians)  $\cong \sin\theta \cong \tan\theta$ )

\* the road gradient in the highway usually does not exceed 4% and on the local roads it could reach 10%-12%.

\* the steepest gradient the vehicle is expected to climb (this may normally be taken as 20%, that is 1 in 5)

#### The gradient effect on the car:

Up hill:

- 1- Increase the car motion resistance;  $F_G = W \sin \theta$  (against the direction of motion)
- 2- Increase the load on rear axle and decrease the load on the front one.
- 3- Decrease the stopping distance when using the brakes (4% is equal to 0.04 g)

Down hill:

- 1- Increase the tractive effort;  $F_G = W \sin \theta$  (in the direction of motion)
- 2- Increase the load on front axle and decrease the load on the rear one.
- 3- Increase the stopping distance when using the brakes (4% is equal to 0.04 g)

# Maximum gradient the car can climb (constant velocity)

#### I-Adhesion

A- Rear wheel drive (neglect the air resistance, and acceleration):

$$\Sigma \mathbf{M}_{A} = 0$$

$$W \cos\theta L_f + W \sin\theta h - R_r L = 0$$
 ....(A-1)

$$\sum \mathbf{F}_{\mathbf{x}} = \mathbf{0}$$

$$F_w = W \sin\theta$$

But 
$$(F_w)_{max} = \mu R_r = W \sin\theta$$

$$R_r = \frac{w \sin \theta}{\mu} \tag{A-2}$$

Substituting Eq. (A-1) into Eq. (A-2)

$$W\cos\theta L_f + W\sin\theta h - \frac{W\sin\theta}{\mu}L = 0 \qquad (A-3)$$

Divide Eq. (3) by (W  $\cos \theta$ )

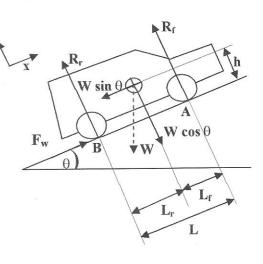
$$L_f + \tan\theta \ h - \tan\theta \ \frac{L}{\mu} = 0$$

$$L_f = \tan\theta \, \frac{L}{\mu} - \tan\theta \, h = \tan\theta \, (\frac{L - h \, \mu}{L})$$

$$\tan \theta = \frac{\mu L_f}{L - \mu h} \tag{A-4}$$

Taking rolling resistance (RR = W  $f_r$ ) into consideration:

$$\tan \theta = \frac{\mu L_f - f_r}{L - \mu h}$$
 (A-5)



### B- Front wheel drive (neglect the air resistance, and acceleration):

$$\Sigma M_B = 0$$

$$W \sin\theta h - W \cos\theta L_r + R_f L = 0$$
 ....(B-1)

$$\sum \mathbf{F}_{\mathbf{x}} = \mathbf{0}$$

$$F_w = W \sin\theta$$

But 
$$(F_w)_{max} = \mu R_f = W \sin\theta$$

$$R_f = \frac{w \sin \theta}{\mu} \tag{B-2}$$

Substituting Eq. (B-1) into Eq. (B-2)

$$W \sin \theta h - W \cos \theta L_r + \frac{W \sin \theta}{\mu} L = 0 \quad \dots \quad (B-3)$$

Divide Eq. (3) by (W  $\cos\theta$ )

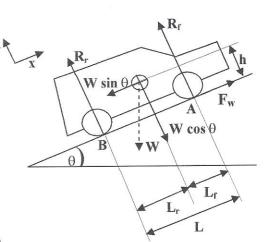
$$\tan\theta \ h - L_r + \tan\theta \ \frac{L}{\mu} = 0$$

$$L_r = \tan\theta \, \frac{L}{\mu} + \tan\theta \, h = \tan\theta \, (\frac{L + h\,\mu}{L})$$

$$\tan \theta = \frac{\mu L_r}{L + \mu h} \tag{B-4}$$

<u>Taking rolling resistance</u> (RR = W  $f_r$ ) into consideration:

$$\tan \theta = \frac{\mu L_r - f_r}{L + \mu h}$$
 (B-5)



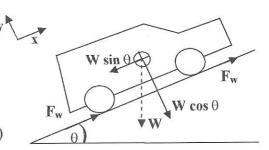
#### C- Four wheel drive (neglect the air resistance, and acceleration):

$$\sum F_x = 0$$

$$F_w = W \sin \theta$$

But  $(F_w)_{max} = \mu W \cos \theta = W \sin \theta$ 

$$\tan \theta = \mu$$
 .....(C-1)



Taking rolling resistance (RR = W  $f_r$ ) into consideration:

$$\tan\theta = \mu - f_r \qquad \qquad \dots \dots (C-2)$$

#### II- Overturn $(R_f = 0)$

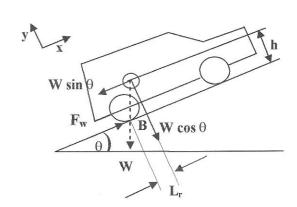
a- Constant speed  $(F_i = 0)$ :

$$\Sigma M_B = 0$$

$$W \sin \theta h - W \cos \theta L_r = 0$$
 ....(II-1)

\* where  $R_f = 0$ 

$$\tan\theta = \frac{L_r}{h} = \frac{L_r/L}{h/L}$$



b- Acclerated car  $(F_i = M a)$ :

The worst condition happens at the first gearbox shift,

at maximum acceleration.  $F_i h = T_w/r$ 

$$\Sigma M_{\rm B} = 0$$

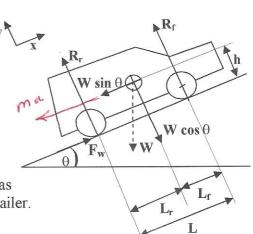
$$T_{w max} i_g i_f h + W sin\theta h - W cos\theta L_r = 0$$

where  $R_f = 0$ 

Overturn condition:

$$\frac{T_{\text{max}} \ i_{\text{g}} \ i_{\text{f}} > W \left( L_{\text{r}} \cos\theta - h \sin\theta \right)}{h} \frac{\omega}{h} \dots \dots (II-1)$$

- \* It is mostly happened to the beach buggy where it has a rear engine (small L<sub>r</sub> and L)
- \* It is also happened to the agricultural tractor where it has a high center of gravity and in case it is coupled to a trailer.



## **III-** Tractive effort

$$\sum F_x = 0$$

$$F_w - W \sin \theta = 0$$
 .....(III-1)

$$\sin \theta = \frac{F_w}{W} \qquad \qquad (\text{III-2})$$

 $W \sin \theta$   $F_w$ 

\* where  $F_w \! < \! \mu \; W$ 

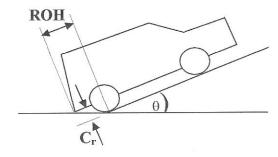
Taking rolling resistance (RR = W  $f_r$ ) into consideration:

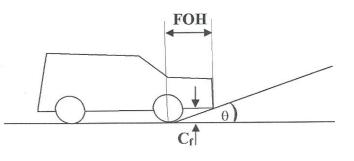
$$\sin \theta = \frac{F_w - W f_r}{W} \qquad \dots \dots (III-3)$$

## IV- Car dimensions

$$\tan \theta = \frac{C_f}{\text{FOH}}$$
, or ..... (IV-1)

$$\tan \theta = \frac{C_r}{\text{ROH}}$$
 (IV-2)





where

FOH = front over hanging  $C_f$  = front under body clearance ROH = rear over hanging  $C_r$  = rear under body clearance

Example:

A car has the following data:

$$m = 1200 \text{ kg}$$
  $i_g = (3.2, 2.8, 1.6, 1)$ 

$$R_w = 30 \text{ mm}$$

$$FOH = 1.3 \text{ m}$$

$$T_{max} = 180 \text{ N.m}$$

$$i_f\!=\!3.8$$

$$C_f = C_r = 0.35 \text{ m}$$

$$ROH = 1.0 \text{ m}$$

$$L = 2.6 \text{ m}$$

$$L_{\rm f} = 1.0 \ {\rm m}$$

 $f_r = 0.015$ 

$$L_{\rm r} = 1.6 \, {\rm m}$$

$$h = 0.6 \text{ m}$$

and

$$\mu = 0.8$$

Find the maximum gradient the car can climb at all possibilities.